



Trane Engineers Newsletter Live Series Energy Saving Opportunities for LEED[®] and the Energy Policy Act

According to the U.S. Green Building Council (USGBC), buildings account for 36 percent of the energy used in the United States. On this ENL broadcast, we will discuss energy-saving strategies to implement for various HVAC system types, and quantify the impact of each toward achieving LEED points under the "Optimize Energy Performance" credit. In addition, we will discuss how these same strategies can help the building owner qualify for tax deductions through the Energy Policy Act.

By attending this event you will:

- 1. Learn energy-saving strategies to implement for various HVAC system types to help earn LEED points.
- 2. Understand the "big picture" of building energy use (impact of building envelope, lighting, plug loads, processes, etc.)
- 3. Learn how energy-saving strategies can help achieve LEED certification nd tax deductions through the Energy Policy Act
- 4. Learn the most common mistakes made during energy modeling and LEED energy analysis projects.

Agenda

- 1) Recent events
 - a) Rising energy costs
 - b) Change to LEED (mandatory two Optimize Energy Performance points)
 - c) Energy Policy Act

2) Energy-saving strategies to achieve two points (energy modeling results, review and explain ECMs)

- a) Rooftop VAV systems
- b) Self-contained VAV systems
- c) Chilled-water VAV systems
- d) Chilled-water fan-coil systems
- e) Water-source heat pump systems
- 3) The "big picture" of building energy use
 - a) Examples
- 4) Top ten energy modeling mistakes
- 5) Summary





Trane Engineers Newsletter Live Series Energy-Saving Opportunities for LEED and the Energy Policy Act (2008)

Matt Biesterveld | C.D.S manager | Trane

Matt is a marketing engineer at Trane with over nine years of experience with HVAC building load and energy analysis. As Manager for the Customer Direct Service (C.D.S.) group, he is responsible for overseeing development, support, and customer training for all C.D.S. HVAC applications.

During his time with C.D.S. he has served various roles as a project manager and later as Team Leader of Design applications. He is also a member of Trane's Advanced Engineering Support (AES) group which is responsible for building load and energy studies including LEED/90.1, EPACT, and performance contracting analyses. He received his B.S. in Mechanical Engineering from the University of Wisconsin – Platteville.

Neil Maldeis | ASC energy engineering manager | Trane

Neil has been with Trane for nine years and is currently responsible for the technical development, support and review of performance based contracting solutions including identifying, verifying, and approval of energy conservation measures on a national basis.

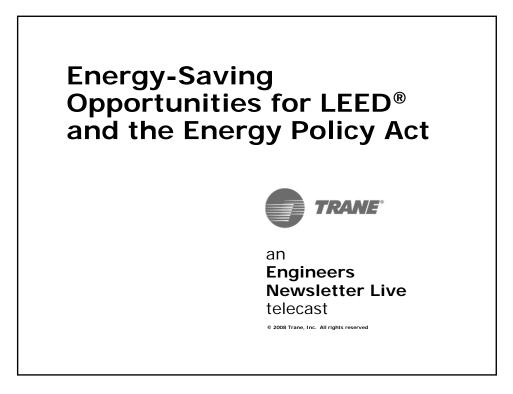
Neil has over 25 years of experience as a mechanical engineer in the building construction and energy conservation fields. He has been responsible for defining/developing the scope of energy conservation solutions, determining the engineering content of the solutions, incorporating customer requirements and analyzing the financial feasibility of the program measures.

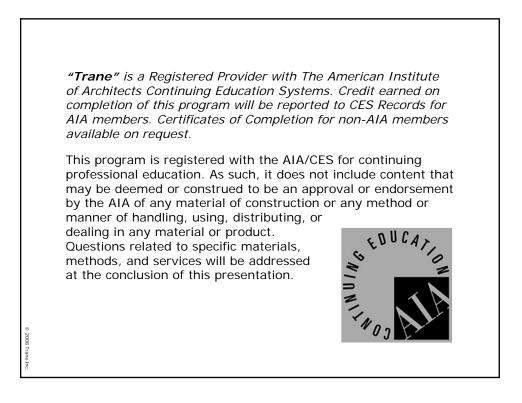
He is a licensed professional engineer in the State of Minnesota and has been certified by the Association of Energy Engineers as Certified Energy Manager and Certified Green Building Engineer. He holds a Bachelors Degree in Mechanical Engineering from the University of Minnesota.

John Murphy | senior applications engineer | Trane

John has been with Trane since 1993. His primary responsibility as an applications engineer is to aid design engineers and Trane sales personnel in the proper design and application of HVAC systems. As a LEED Accredited Professional, he has helped our customers and local offices on a wide range of LEED projects. His main areas of expertise include dehumidification, air-to-air energy recovery, psychrometry, ventilation, and ASHRAE Standards 15, 62.1, and 90.1.

John is the author of numerous Trane application manuals and Engineers Newsletters, and is a frequent presenter on Trane's Engineers Newsletter Live series of satellite broadcasts. He also is a member of ASHRAE, has authored several articles for the ASHRAE Journal, and is a member of ASHRAE's "Moisture Management in Buildings" and "Mechanical Dehumidifiers" technical committees.

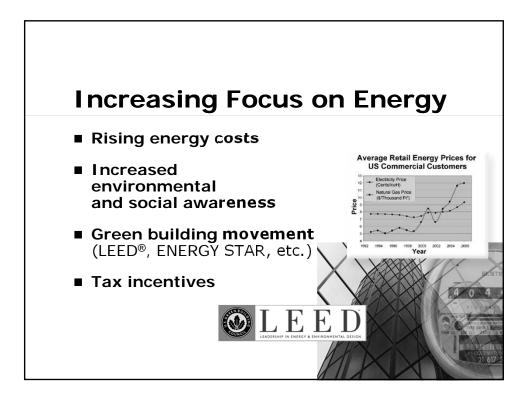


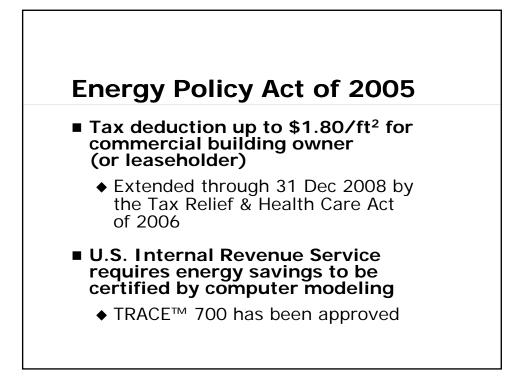


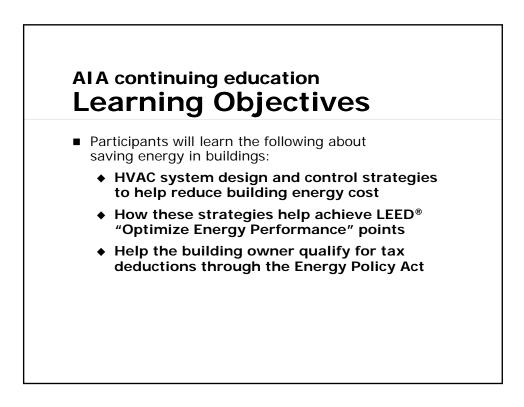


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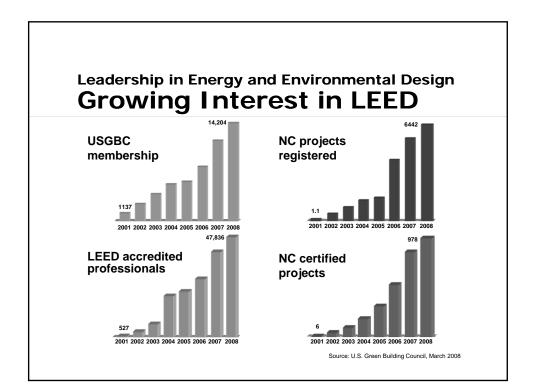


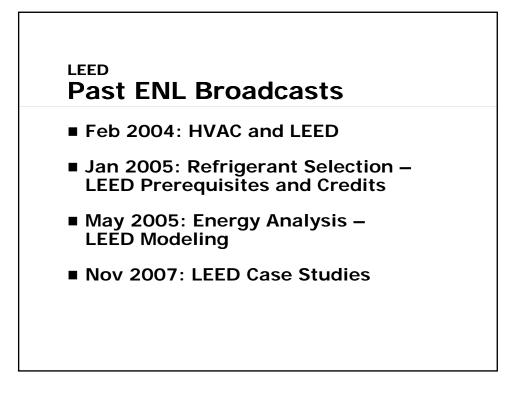


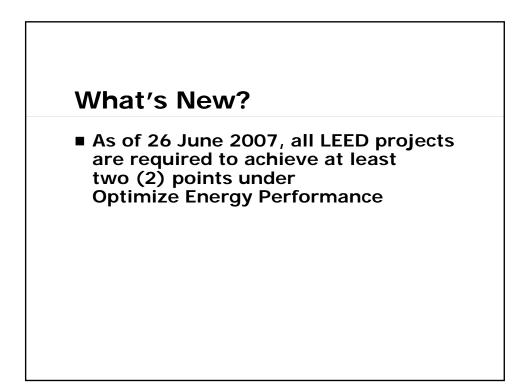




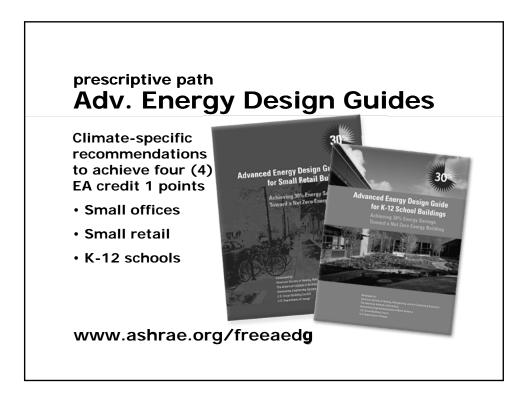
Energy-saving opportunities for LEED[®] and the Energy Policy Act

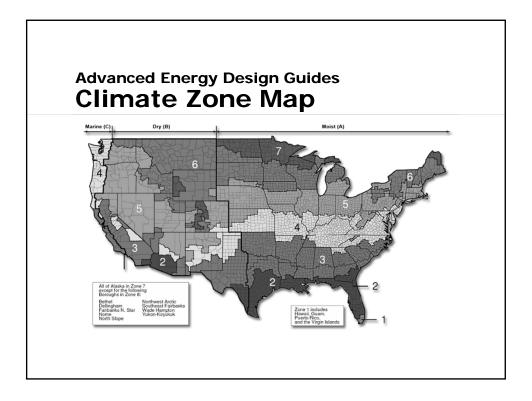




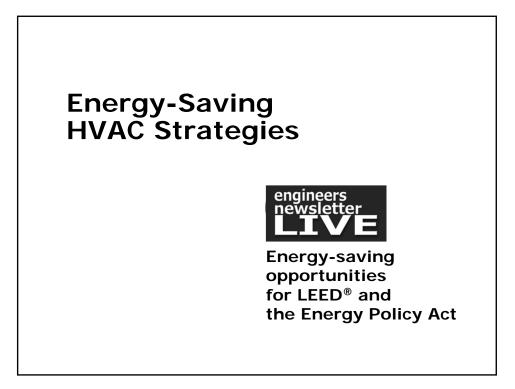


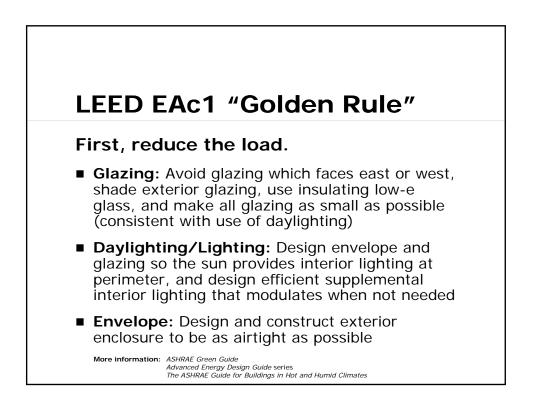
| EA Credi | on 2.2 t 1 | | |
|--|----------------------|---------------------|----------------|
| | New building | Major renovation | LEED points |
| Reduction of | 10.5% | 3.5% | 1 |
| overall building energy cost compared to a | 14 | 7 | 2 |
| | 17.5 | 10.5 | 3 |
| baseline building | 21 | 14 | 4 |
| defined by ASHRAE 90.1-2004 | 24.5 | 17.5 | 5 |
| (Appendix G) | 28 | 21 | 6 |
| | 31.5 | 24.5 | 7 |
| | 35 | 28 | 8 |
| | 38.5 | 31.5 | 9 |
| | 42 | 35 | 10 |

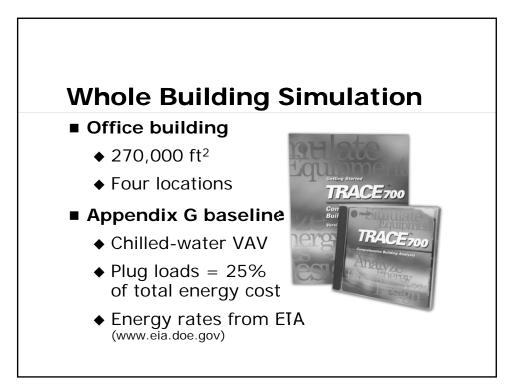




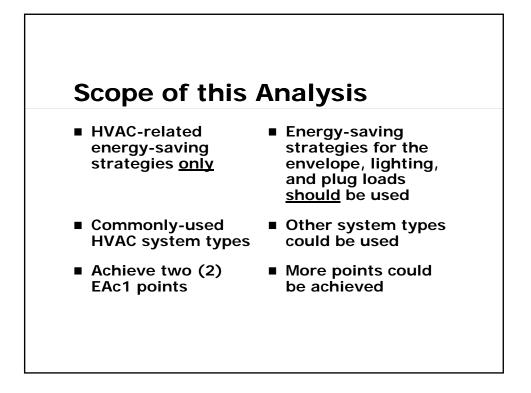
| | | | ergy Des | - | | | |
|----------|---------|-----------------------------------|--|----------------|---|--------------------------------|-------------|
| | | | | - | | | |
| | 201 | | | | | \mathbf{b} | |
| - | | | nenaa | atio | ons Ta | | |
| | lte | em | Component | | Recommendation | How | -To Tip |
| | | | n entirely above deck | R-25 c.i | i. | EN1-2 | |
| | Roofs | Attic and | | R-30 | | EN3, EN15-16 | |
| | | Metal bu | ilding | R-19 | | EN3-4 EN15, E | EN18 |
| | | SRI | | 0.78 | | EN1 | 14.0 |
| | | | C > 7 Btu/ft ^{2,} °F) | R-5.7 c. | .1. | EN5, EN15, EN | |
| | Walls | Steel fra | med amed and other | R-13 R-13 | | EN6, EN15, EN EN7, EN15, EN | |
| | vvans | Metal bu | | R-13 R-16 | | EN7, EN15, EN | |
| | | | ade walls | | with Standard 90.1* | EN8, EN15, EN | |
| e | | Mass | | R-4.2 c | | EN9, EN15, EN | |
| l de | Floors | Item | | R-19 | | EN10, EN15, E | N18 |
| Envelope | | Item | Compone | | Becomm | endation | |
| | Slabs | | Heat pump (≥65 and < | 135 kBtu/h) | 10.6 EER/3.2 COP | Iondation | Но |
| | | | Heat pump (≥135 kBtu | /h) | 10.1 EER/11.5 IPLV/3.1 C | OP | |
| | Doors | Packaged DX | Gas furnace (<225 kBl Gas furnace (≥225 kBt | tu/h) | 80% AFUE or E, | | HV1, HV7- |
| | | Rooftops (or DX Split Systems) | Economizer | u/n) | 80% E _c | | |
| | | opin oysterns) | Ventilation | | Comply with Standard 90. | 1* | HV13 |
| | Vertica | | Fans | | Energy recovery or deman | nd control | HV9, HV11- |
| | Fenest | | | | Constant volume: 1 hp/100 Variable volume: 1.3 hp/10 | 00 cfm | HV19 |
| | | | Water-source heat | | Cooling: 12.0 EER at 86°F | JUU CIM | 11119 |
| | | | pump (<65 kBtu/h) | | Heating: 4.5 COP at 68°F | | |
| | Interio | | Water-source heat pum | p (≥65 kBtu/h) | Cooling: 12.0 EER at 86°F | | HV2, HV7-8 |
| | | | Ground-source heat put | mn | Heating: 4.2 COP at 68°F | | |
| | | | (GSHP) (<65 kBtu/h) | | Cooling: 14.1 EER at 77°F Heating: 3.5 COP at 32°F a | and 17.0 EER at 59°F | |
| | | WSHP System | GSHP (≥65 kBtu/h) | | Cooling: 13.0 EER at 77°F Heating: 3.1 COP at 32°F a | and 16 0 EED at some | HV2, HV7-8 |
| | | | Gas boiler | | 85% E | ind 3.5 COP at 50°F | |
| | | | Economizer | | OU NO L | | HV2, HV7, H |

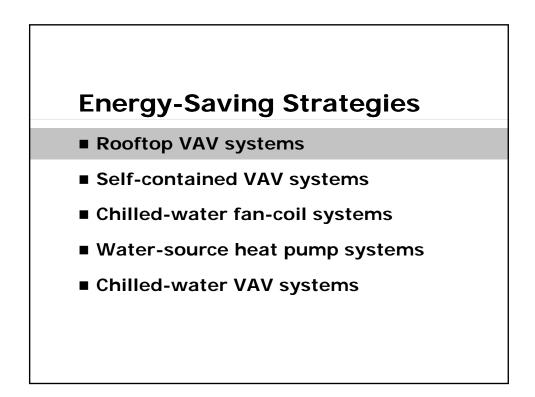


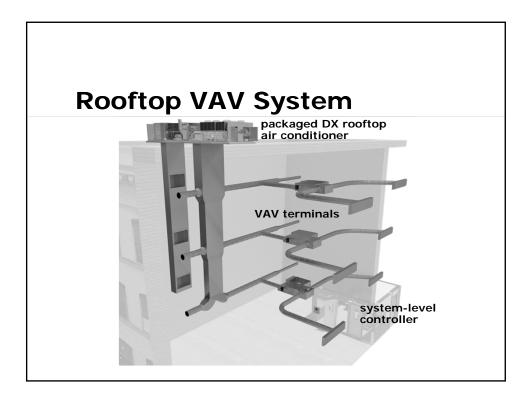


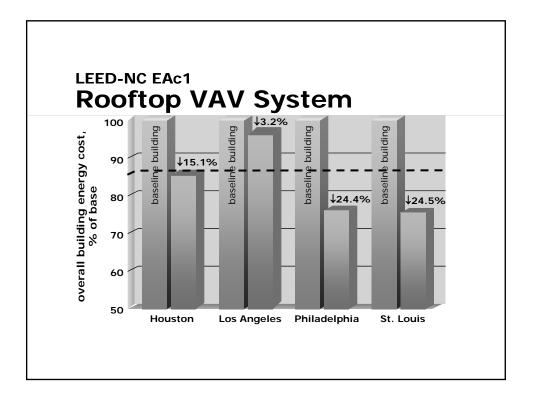




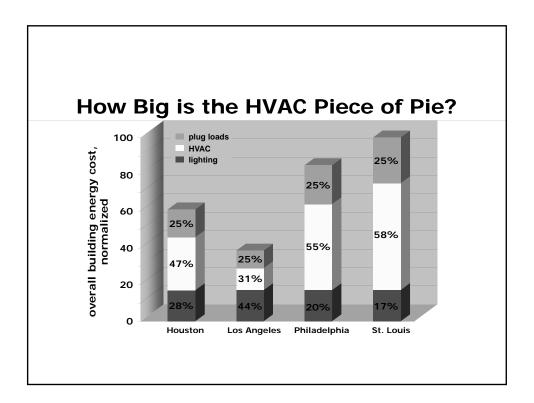


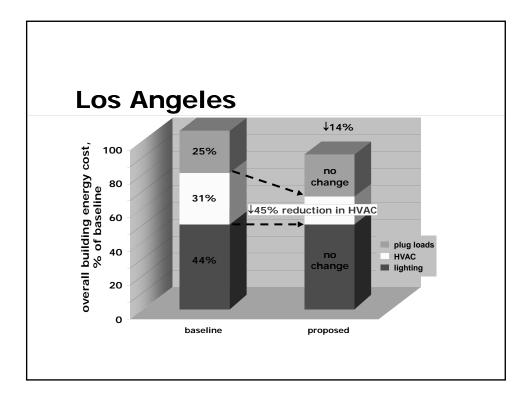




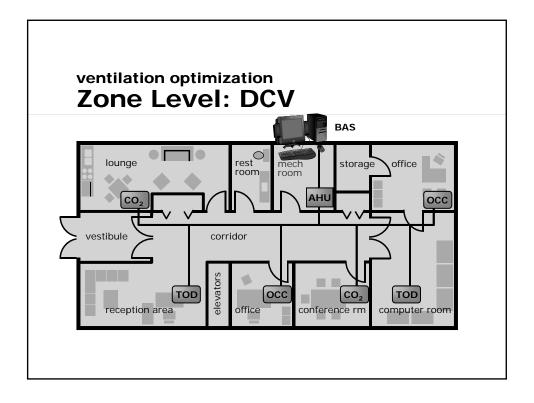


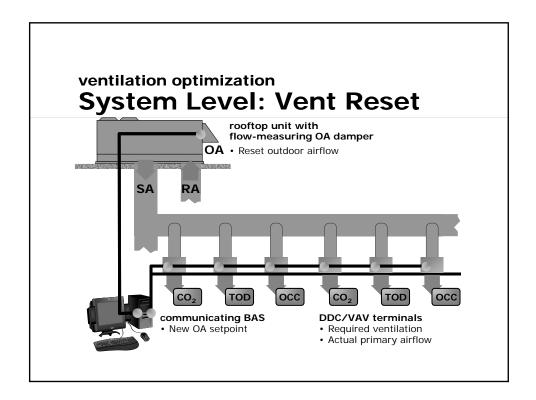
| norav covina ct | ratagi | | | |
|---|---------|-------------|--------------|-----------|
| nergy-saving st Rooftop V | - | | n | |
| | | yster | | |
| | Houston | Los Angeles | Philadelphia | St. Louis |
| High-efficiency rooftop unit (9.7 EER) | x | x | x | x |
| Ventilation optimization (demand-controlled ventilation at zone-level + ventilation reset at | x | x | x | x |
| system level) Improved supply-air- | | | | |
| temperature reset | х | x | x | x |
| Parallel-fan-powered VAV (serving perimeter zones) | x | x | x | x |
| Cold-air distribution (52°F supply-air + 1°F increase on space cooling setpoint) | x | x | | |
| Airside economizer | x | x | х | x |

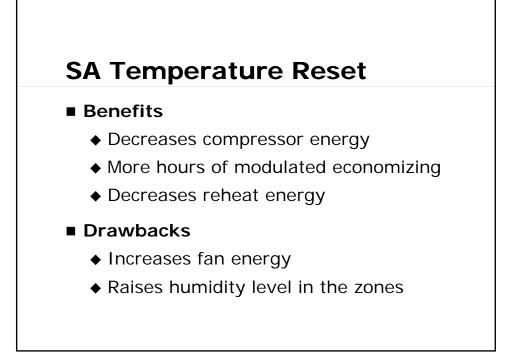


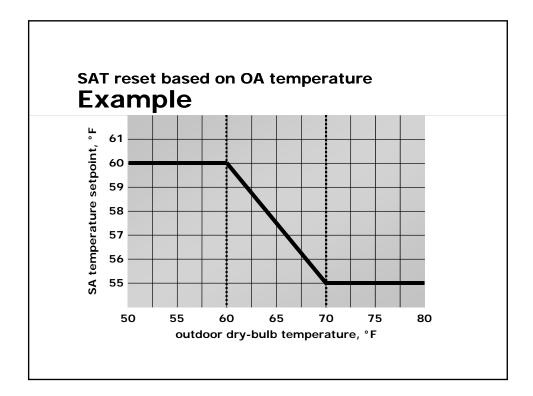


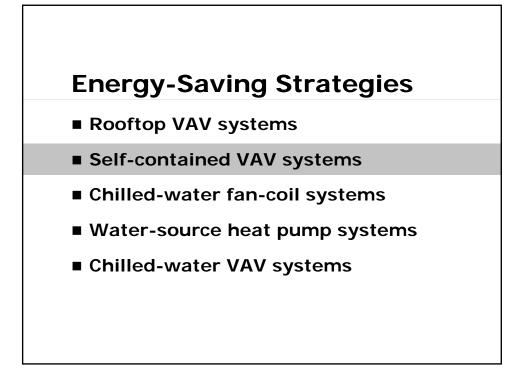
| energy-saving st | rategie | 25 | | |
|--|---------|----|--------------|-----------|
| Rooftop V | - | | n | |
| | Houston | • | Philadelphia | St. Louis |
| High-efficiency rooftop unit (9.7 EER) | X | X | X | X |
| Ventilation optimization (demand-controlled ventilation at zone-level + ventilation reset at system level) | × | x | х | x |
| Improved supply-air- temperature reset | x | x | x | х |
| Parallel-fan-powered VAV (serving perimeter zones) | x | x | x | х |
| Cold-air distribution (52°F supply-air + 1°F increase on space cooling setpoint) | x | х | | |
| Airside economizer | x | x | x | х |

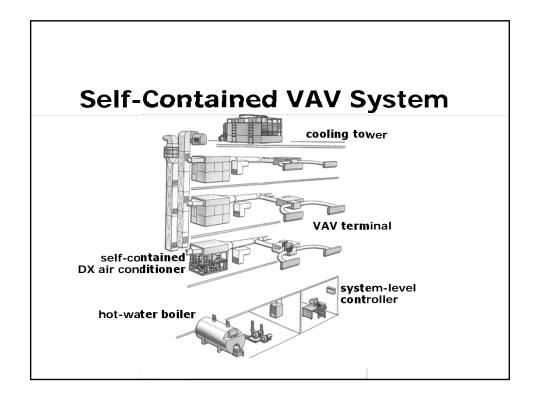


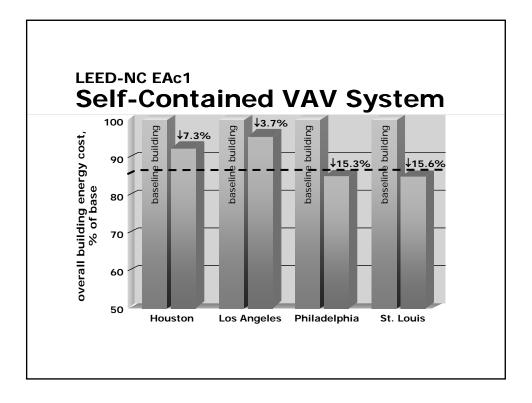




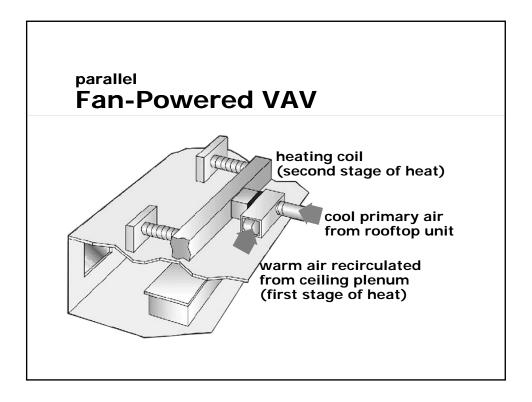


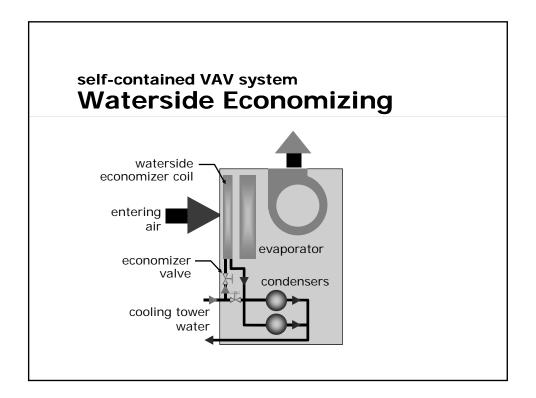


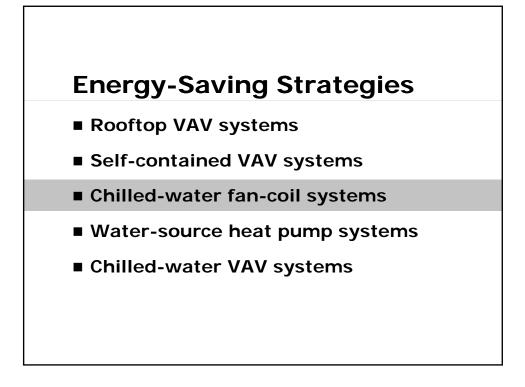


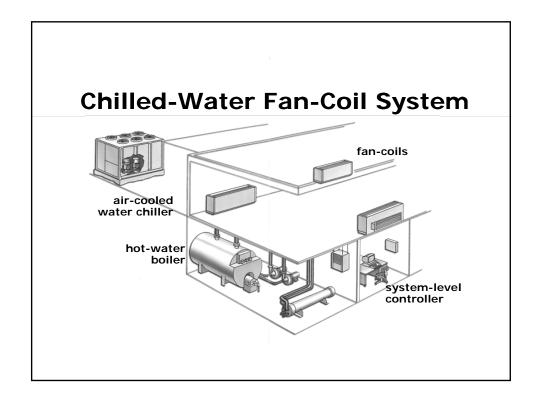


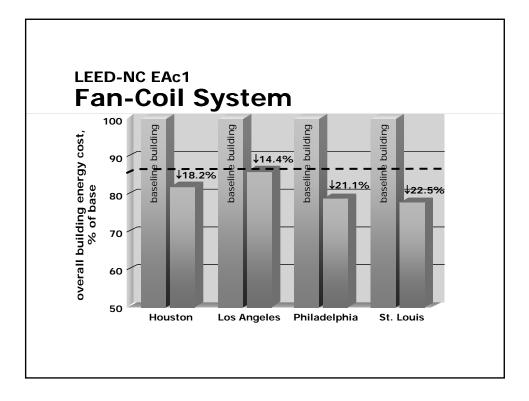
| nergy-saving st | | | | |
|--|---------|-------------|--------------|-----------|
| Self-Conta | inec | I VAV | Syst | em |
| | Houston | Los Angeles | Philadelphia | St. Louis |
| Ventilation optimization (demand-controlled ventilation at zone-level + ventilation reset at system level) | x | x | x | х |
| Improved supply-air- temperature reset | x | x | x | х |
| Lower condenser flow rate (12°F ΔT) | х | х | х | х |
| Parallel-fan-powered VAV (serving perimeter zones) | x | x | x | x |
| Cold-air distribution (50°F supply-air + 1°F increase in space cooling setpoint) | x | х | | |
| Exhaust-air energy recovery (total-energy wheel) | x | | | |
| Waterside economizer | х | х | х | х |
| Optimized tower control | | Х | | |



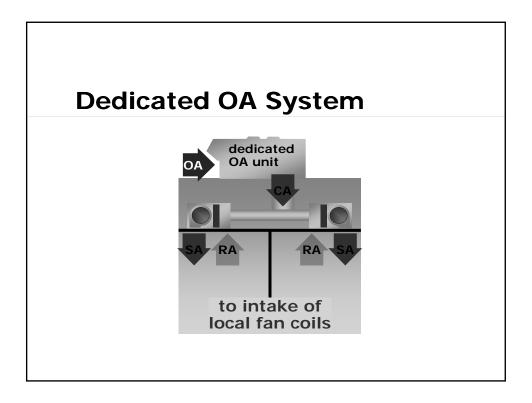


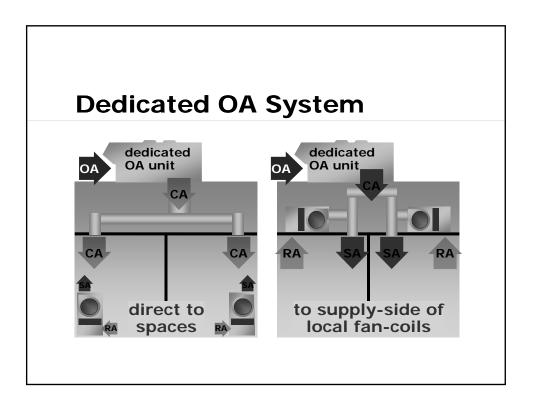


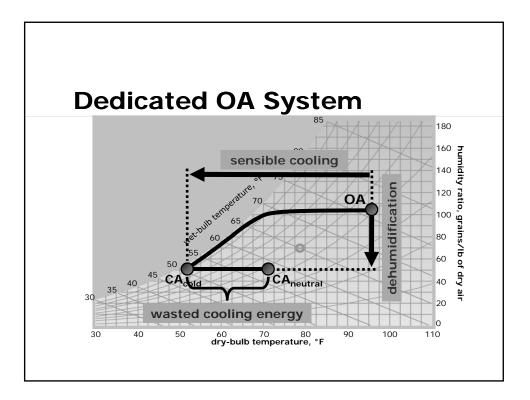


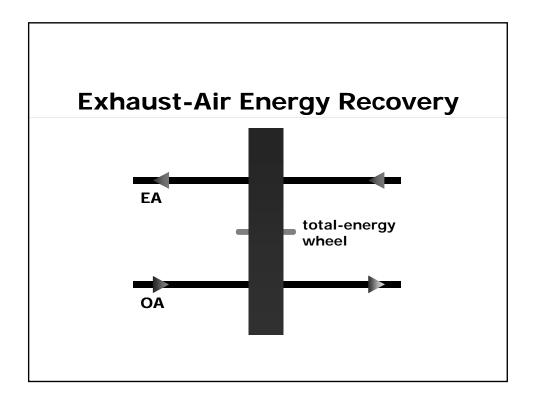


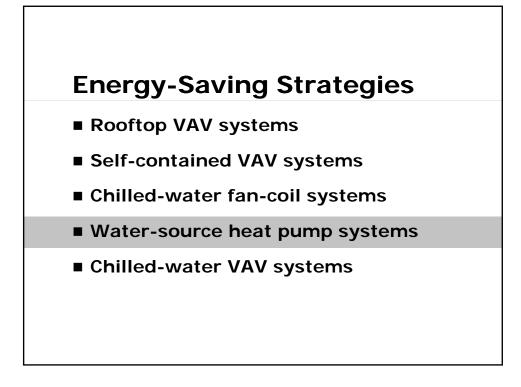
| energy-saving str | ategie | es | | |
|--|---------|-------------|--------------|-----------|
| | - | | | |
| Fan-Coil Sy | /216 | | | |
| | Houston | Los Angeles | Philadelphia | St. Louis |
| Lower condenser flow rate (15°F ΔT) | х | x | x | x |
| Lower evaporator flow rate (14°F Δ T) | х | х | x | x |
| High-efficiency, water-cooled centrifugal chiller (0.59 kW/ton, or 0.48 kW/ton for Los Angeles) | x | x | x | x |
| Multiple-speed fans in fan coils | s X | х | х | х |
| Deliver conditioned OA cold (rather than "neutral") directly to spaces | УХ | x | x | x |
| Demand-controlled ventilation | | х | | |
| Exhaust-air energy recovery (total-energy wheel in dedicated OA unit) | x | | x | x |
| Waterside economizer (plate- and-frame heat exchanger) | | х | x | х |
| Chiller-tower optimization control | х | х | Х | х |

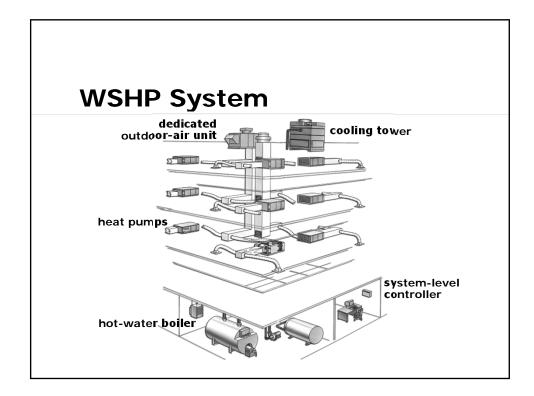


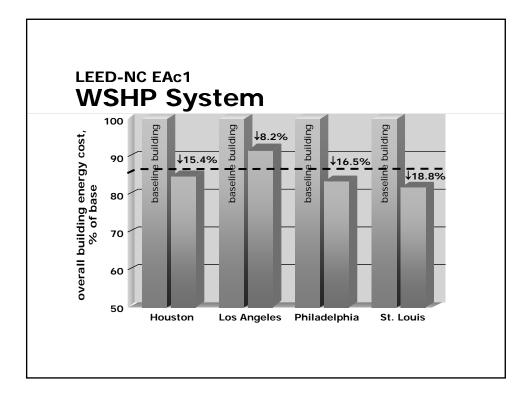




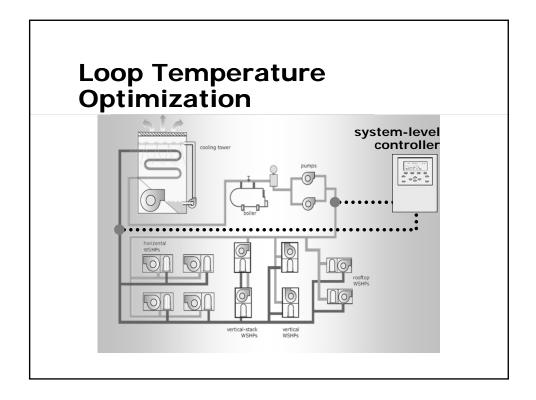


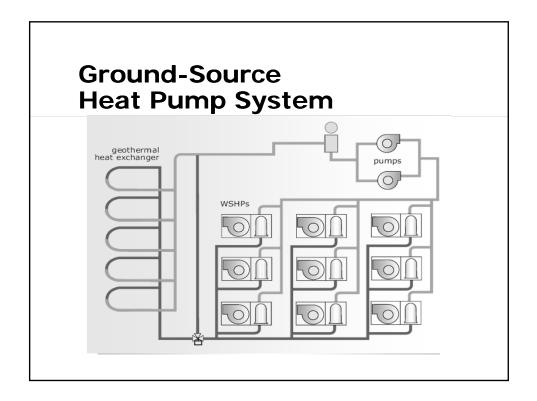


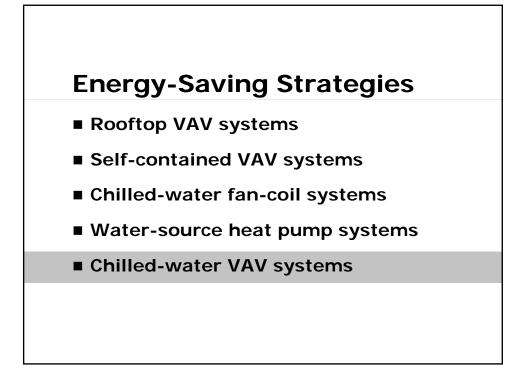


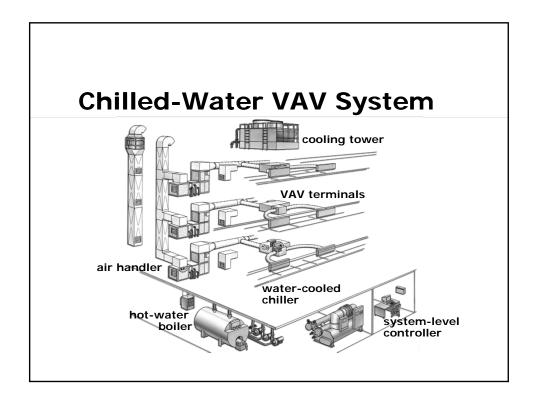


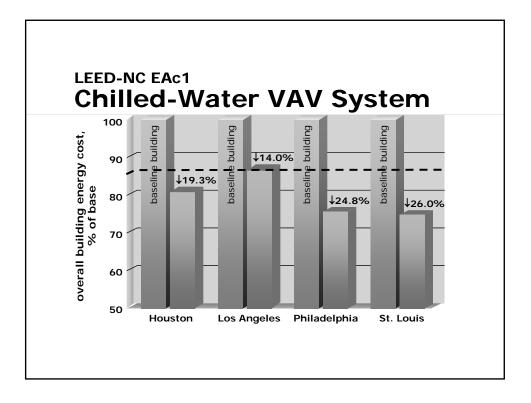
| energy-saving st | ratodia | 26 | | |
|---|---------|-------------|--------------|-----------|
| WSHP Sys | | -3 | | |
| | Houston | Los Angeles | Philadelphia | St. Louis |
| High-efficiency water- source heat pumps (15.7 EER) | x | x | x | x |
| Loop temperature optimization control sequence | x | х | х | x |
| Deliver conditioned OA cold (rather than "neutral") to spaces | x | х | x | x |
| Cycle WSHP fans with load | | х | | |
| Exhaust-air energy recovery (total-energy wheel in dedicated OA unit) | х | | x | х |
| Demand-controlled ventilation | | x | | |
| Waterside economizer | | х | | |



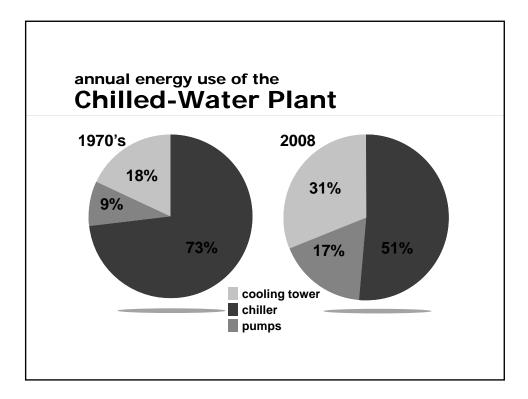




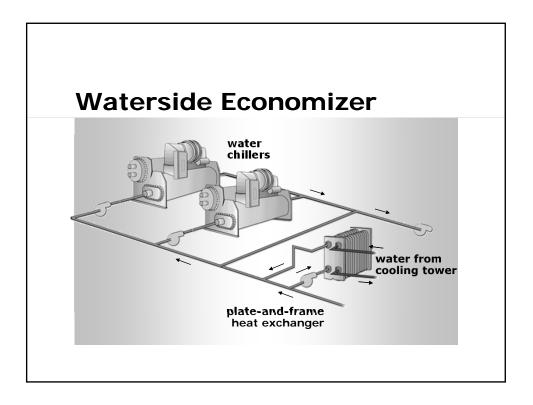


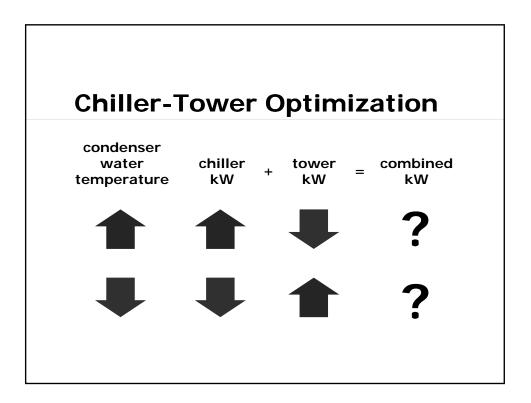


| energy-saving strat Chilled-Wate | - | | Svste | m |
|---|--------|-------------|-------|---|
| | ouston | Los Angeles | | |
| Lower condenser flow rate (15°F Δ T) | х | x | x | х |
| Lower evaporator flow rate (14°F Δ T) | х | х | х | х |
| High-efficiency, water-cooled centrifugal chiller (0.59 kW/ton, or 0.48 kW/ton for Los Angeles) | х | x | х | x |
| Ventilation optimization (DCV at zone-level + ventilation reset at system level) | x | x | x | x |
| Improved supply-air-temperature reset | х | х | x | х |
| Parallel fan-powered VAV (serving perimeter zones) | | х | x | x |
| Cold-air distribution (48°F supply-air + 1°F increase in space cooling setpoint) | | x | x | x |
| High-efficiency, airfoil supply fan | х | х | х | х |
| Airside economizer | | х | | |
| Chiller-tower optimization | Х | Х | Х | х |



| c | hilled / conde | nser flow rates (gp | m/to |
|---------------------|----------------|---------------------|------|
| | 2.4 / 3.0 | 1.5 / 2.0 | |
| chiller | 256 kW | 292 kW | |
| chilled-water pump | 32 | 8 | |
| condenser-water pum | р 25 | 9 | |
| cooling tower fans | 24 | 16 | |
| total kW | 337 | 325 | |





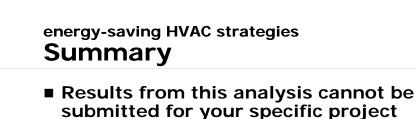
"Cold" Supply Air

Benefits

- Reduces supply airflow (smaller air handlers, ductwork, VAV terminals, and supply fan)
- ◆ Reduces supply fan energy

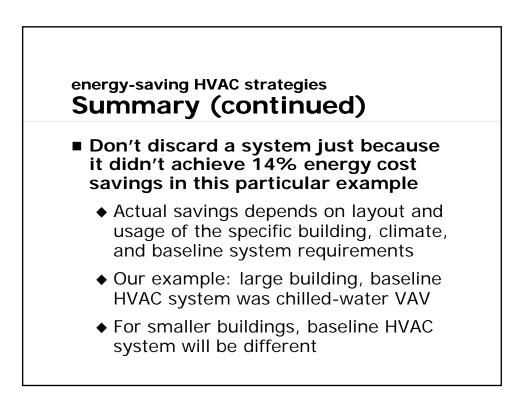
Drawbacks

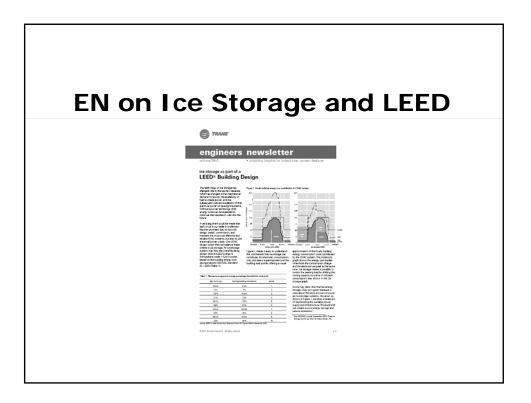
- Increases reheat energy
- Fewer hours when economizer provides all necessary cooling (compressors shut off)

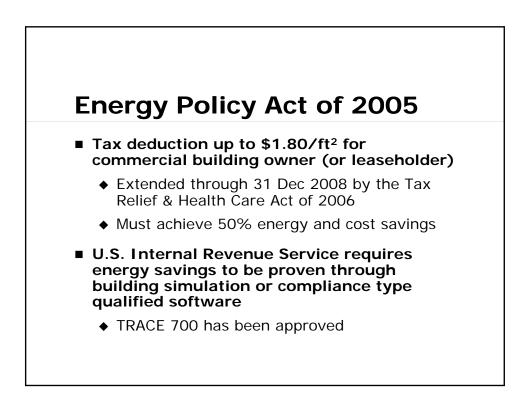


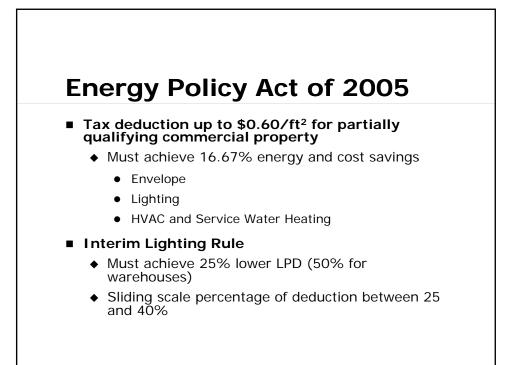
- USGBC requires the actual building to be modeled to estimate savings
- Goal: To provide guidance as to what strategies should be investigated for the HVAC system you may be leaning toward using on a given project

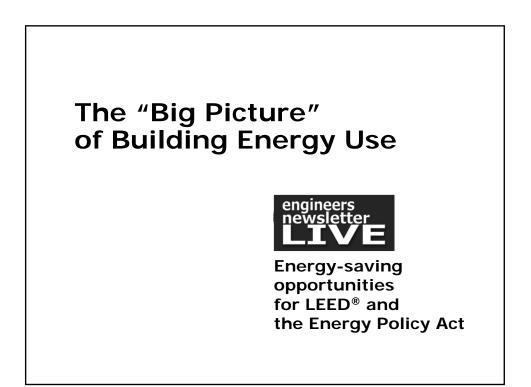


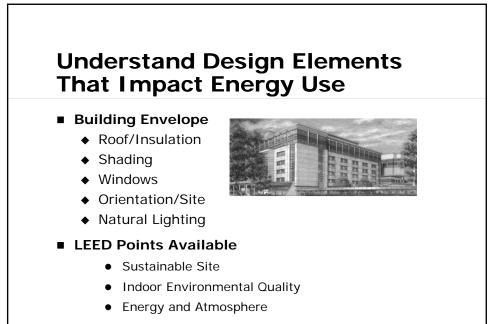


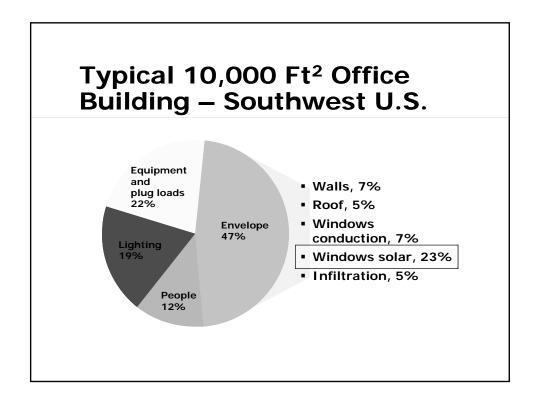












| Window U-factors, ti by the building | nting or | films, s | hading | provided |
|---|-----------------------------------|---------------------|-----------------------------------|--------------------------------|
| by the building | Typical window performance values | | | |
| Glazing type | U-value of glazing | Shading coefficient | Solar heat gain coefficient | Visible light transmittance |
| Single-pane, clear | 0.88 | 1 | 0.86 | 90% |
| Double-pane, clear | 0.48 | 0.87 | 0.75 | 81% |
| Double-pane, clear, low-e | 0.32 | 0.7 | 0.6 | 73% |
| Double-pane, tinted (bronze) | 0.48 | 0.59 | 0.5 | 48% |
| Double-pane, tinted (green), low-e | 0.32 | 0.48 | 0.42 | 61% |
| Double-pane, reflective | 0.48 | 0.26 | 0.22 | 18% |

| Typical 10,000 ft ² office building in Sou | thwestern U. |
|---|--------------------|
| Window type | Percent savings |
| Single-pane, clear | 0% |
| Double-pane, clear | 5% |
| Double-pane, clear, with shade screen (SC=0.35) | 20% |
| Double-pane, tint (SC=0.50), low-e | 11% |
| Double-pane, reflective (SC=0.20) | 26% |

Impact of Insulation and **Roof Color**

Typical 10,000 ft² office building in Southwestern U.S. **Insulation** level Roof color Cost ft² dark \$2.26 No insulation R=0

| R-19 | | |
|------|--|--|
| R-30 | | |

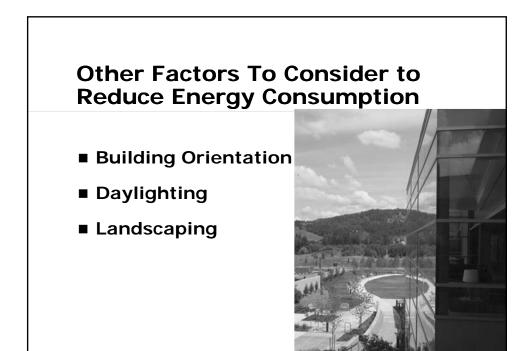
| light | \$ 1.69 |
|-------|---------|
| dark | \$ 1.23 |
| light | \$ 1.11 |

\$ 2.17

\$ 1.80

light

dark





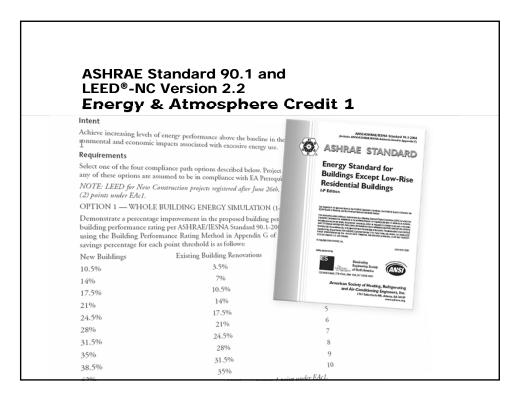


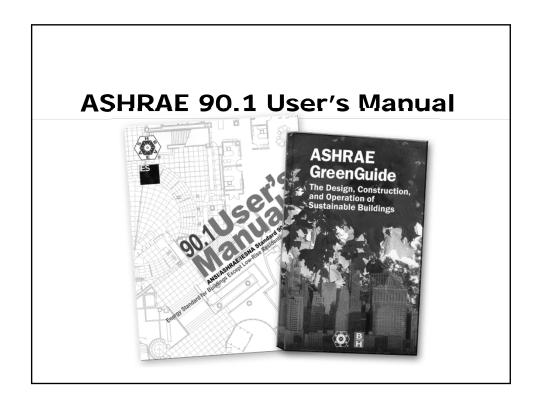








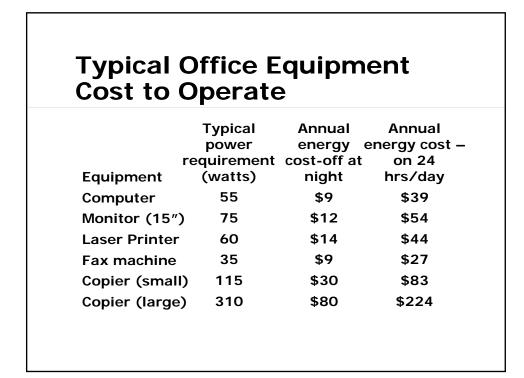


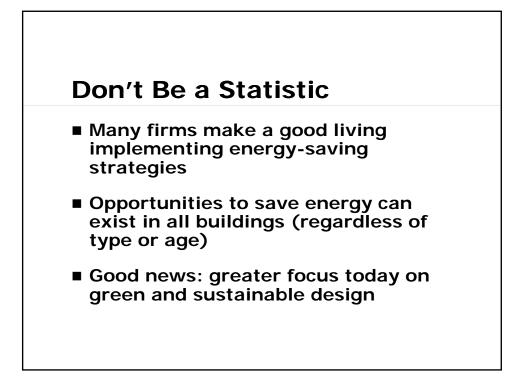


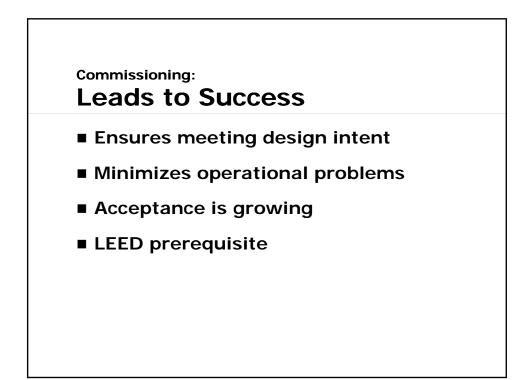
Consider Use Factors That Impact Building Energy Use

Internal Loads

- Equipment loads
- Lighting systems
- Operating Schedules
 - HVAC schedules
 - Thermostat settings and set back
- Occupancy Schedules
 - People loads

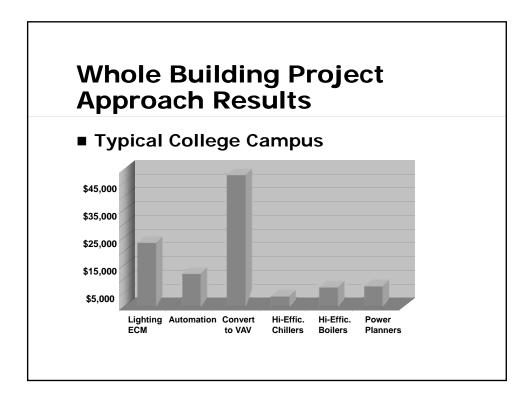


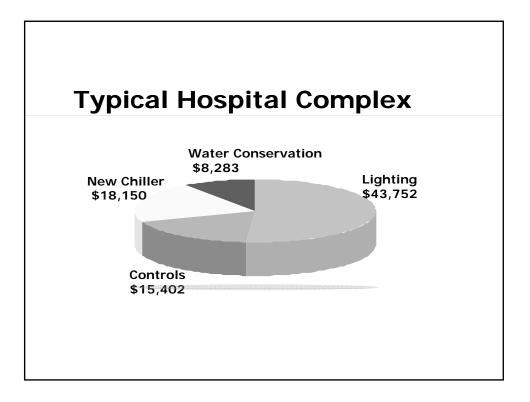


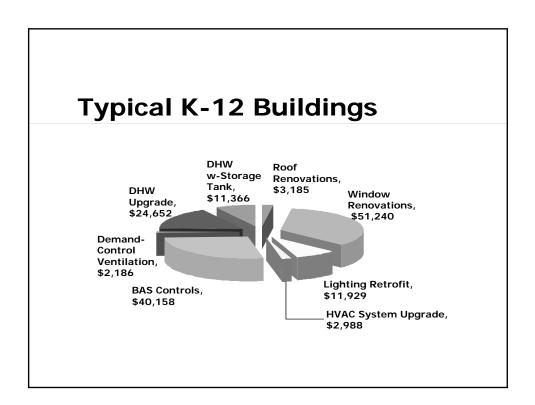




- Alternatives are easy to set up
- Do "What ifs" to see energy impact
- Simple changes or improvements can have a huge impact
- Modeling automatically takes a whole building approach







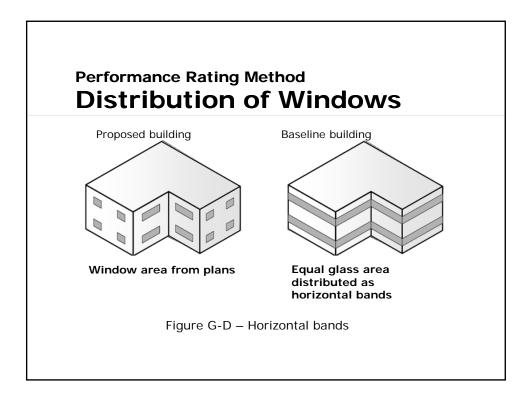


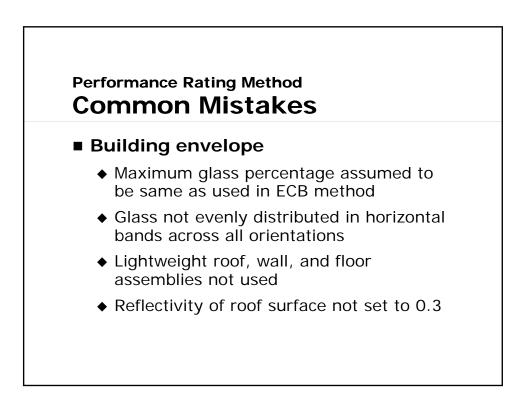


Energy-saving opportunities for LEED[®] and the Energy Policy Act

Performance Rating Method Common Mistakes Building envelope Maximum glass percentage assumed to be same as used in ECB method

 Glass not evenly distributed in horizontal bands across all orientations





Performance Rating Method Common Mistakes

HVAC system

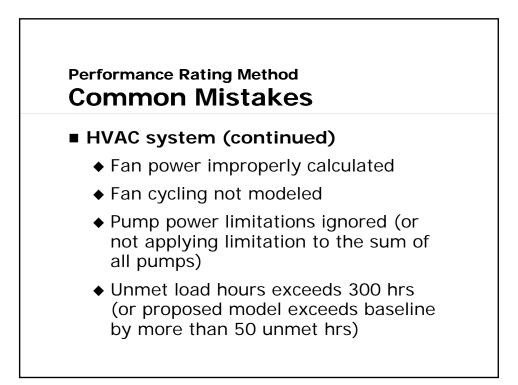
Incorrect baseline system selection

Performance Rating Method Baseline HVAC Systems Table G-A-Baseline Building HVAC System Types and Descriptions This table combines Table G3.1.1.4 and Table G3.1.1B from Standard 90.1-2004 for ease of use and clarity. The building types and sizes are shown as columns. System types and specifications are shown in rows. These are grouped first by proposed buildings that use fossil fuels and then by proposed buildings that use electricity. Nonresidential 4 or 5 floors or less than 75,000 ft² or Less than 3 floors or less 5 floors or less and More than 5 floors or Residential Heating Source More than 3 stories than 75,000 ft² 75,000–150,000 ft² more than 150,000 ft² Fossil Fuel, Number/Code 1-PTAC 3-PSZ-AC 5—PVAV w/Reheat 7—VAV w/Reheat Fossil/Electric Hybrid & System Type Packaged terminal air Packaged rooftop air Variable air volume with Packaged 100ftop Purchased Heat variable air volume with reheat conditioner conditioner reheat Constant Volume Fan Control Constant Volume VAV VAV Chilled Water Cooling Type Direct Expansion Direct Expansion Direct Expansion Heating Type Hot Water Fossil Fuel Fossil Fuel Furnace Hot Water Fossil Fuel Hot Water Fossil Fuel Boiler Boiler Boiler Electric and Other Number/Code 2-PTHP 4—PSZ-HP 6-PVAV w/ PFP 8-VAV w/ PFP Boxes Boxes System Type Packaged terminal heat Packaged rooftop heat Packaged 100ftop Variable air volume with variable air volume with pump pump reheat reheat Constant Volume Fan Control Constant Volume VAV VAV Chilled Water Cooling Type Direct Expansion Direct Expansion Direct Expansion Heating Type Electric Heat Pump Electric Heat Pump Electric Resistance Electric Resistance

Performance Rating Method Common Mistakes

HVAC system

- Incorrect baseline system selection
- Packaged unit supply fan, condenser fan, compressor energy not properly separated
- ◆ IPLV requirement ignored
- Use of default equipment that significantly exceeds (or does not meet) required part load performance



Performance Rating Method Common Mistakes

HVAC system (continued)

- Optimum start ignored for systems greater than 10,000 cfm
- ◆ Design flow rates not using a 20°F ∆T for baseline building systems
- ♦ 8760 hour simulation not used

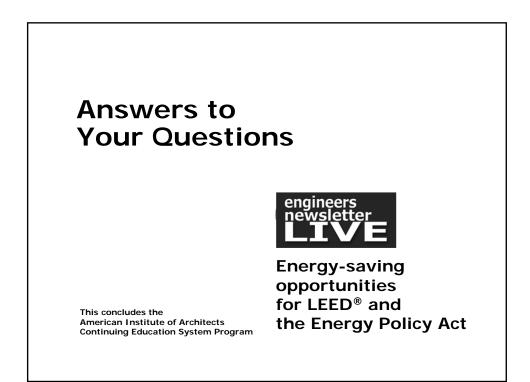
Energy Modeling Common Mistakes

- General
 - Using default or invalid assumptions on schedules
 - Using default equipment despite significant changes in design conditions
 - Over simplifying thermal zoning
 - Improperly entering ventilation



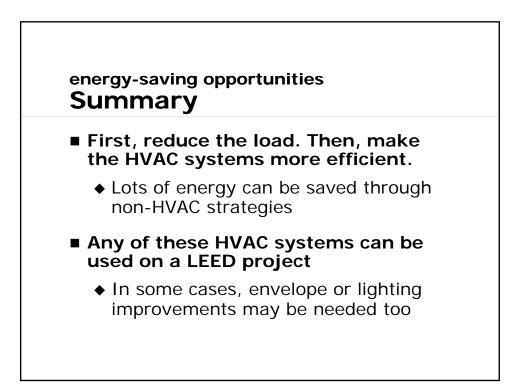
General (continued)

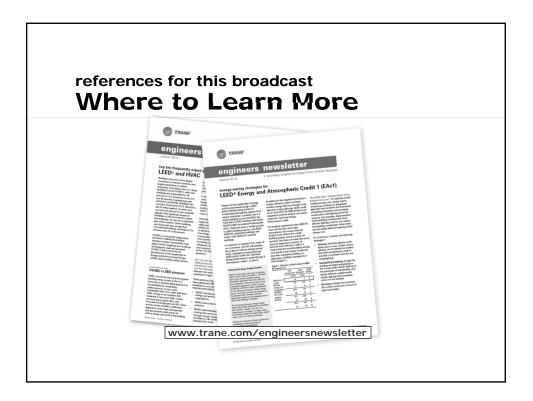
- Overly constraining the model to meet expectations
- Extending modeling capabilities beyond the intent
- Using design assumptions during typical building simulation

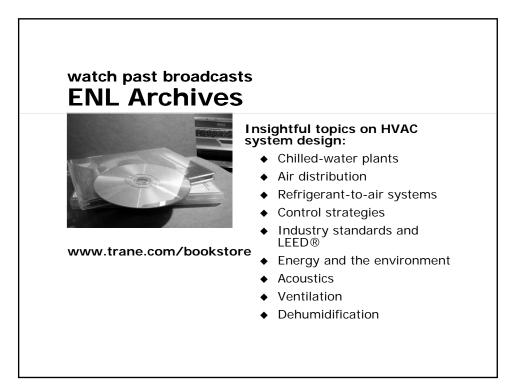




- Impact of energy-saving strategies depend on climate, building usage, and utility costs
- Building analysis tools (like TRACE[™]) can convert energy savings to operating cost dollars to:
 - Save clients money
 - Achieve LEED points under EA credit 1
 - Qualify for tax deductions (Energy Policy Act)









- September 10 Small chilled-water systems
- November 12 ASHRAE Standards 90.1 and 62.1 and VAV Systems